MSC MARC finite element analysis software usage course

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Presentation: Alexandra Kemény





Course data

- 5×3 h course
- 12 participants
- Online platform via MS Teams
- Finite element analysis basics
- MSC MARC basics
- Material properties
- Forming and welding processes





Part 1





Installing and licensing the software (Marc Mentat 2019 Feature Pack 1)







MSC Marc units

| | N-mm-s-K | N-m-s-K (SI) |
|-------------------------------|--|--------------------|
| force | 1 N | 1 N |
| distance | 1000 mm | 1m |
| time | 1s | 1s |
| mass | 1.10 ⁻³ Mg | 1 kg |
| stress | 1.10 ⁻⁶ MPa | 1 Pa |
| energy, work | 1.10 ³ Nmm | 1 J (Nm) |
| Young's modulus | 1.10 ⁻⁶ MPa | 1 Pa |
| density | 1.10^{-12} Mg/mm ³ | 1 kg/m^3 |
| conductivity (K) | 1Nmm/(mmK) | 1W/(mK) |
| specific heat | $1.10^{6} \text{ mm}^{2}/(\text{s}^{2} \text{ K})$ | 1 J/(kg K) |
| film coefficient (face film) | 1.10^{-3} Nmm/(mm ² K) | $1 W/(m^2 K)$ |
| latent heat | 1.10 ⁶ Nmm/Mg | 1J/kg |
| thermal expansion coefficient | 11/K | 11/K |







MSC Marc stress-strain calculations

| | | | Stress | Strain |
|--------------|------------------|------------------------------|------------------------|--------------------|
| Small strain | _ | | Engin. | Engin. |
| Small Strain | Large ro | tation | Engin. | Engin. |
| Large Strain | Total Lag | grange | II. Piola- Kirchhof | Green- Lagrange |
| | Updated Lagrange | Adaptive decomposition | True (Cauchy) | True |
| | | Multiplicative decomposition | True (Cauchy) | True |





Basics

- Overview of the program
 - Plot control visible controls settings
- Creating an element
 - First creating the nodes, then adding element
 - Import from 3D model
- Mesh refinement
 - Geometry and Mesh operations subdivide
- Platin strain 2D geometry with thickness
- Job
 - Job results settings
 - Output file error message decoding





Exercise 1&2 Deforming a simple object





Exercise 1

- Fixed x=0 displacement on bottom
- Fixed y=0 displacement from left bottom
- Fixed y=0.02 displacement on top sides











Exercise 2

- Fixed x=0 displacement on bottom
- Fixed y=0 displacement from left bottom
- Fixed y=0.02 displacement on top line











Exercise 3&4 - Beam Explained step-by-step





Exercise 3

- Fixed x=0, y=0 and z=0 displacement on left side
- Point load (y direction) on right end



































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Exercise 4

- Fixed x=0, y=0 and z=0 displacement on left side
- Point load (y direction) on right end
- Mesh refinement for better simulation results









Homework 1





Homework 1

- Simulate the pressure effect on the tube.
- Use line element, expand and edge load commands.









Part 2





Tips and tricks for the exercises of Part 2





- Selection control: set edge flood to 60°. If the angle between 2 edges are smaller than the set value, when 1 is selected, both are.
- Multi-Criteria stepping: the software sets the time steps, if the change is fast, the steps become shorter, if the change is slow, the steps become longer.
- If anything goes wrong, hit Check.
- Automatic meshing: tri mesh and tetra mesh can mesh anything, but a little more inaccurately than quad mesh. However, quad mesh is not always applicable.







- For 3D models, it is recommended to use a different software, Hypermesh, but Marc can also mesh the models about fine.
- Plain stress mode: no stress in direction Z.
- Plain strain mode: no strain in direction Z.
- If near a stress concentration, a high stress is simulated, it is only prompt if more than one element has the same stress level.
- When importing an igs file, the 2 bottom boxes should be checked.





- Friction can be added in contacts contact interactions.
- Separation: it is used when 2 surfaces are in contact, defines the needed stress value to separate them (glued surfaces).
- If displacement/speed is set in contacts, it is not needed to fix the geometric features with boundary conditions
- Job: Contact Control:
 - Node to segment the values are only calculated in the nodes
 - Segment to segment calulate the segments of the meshed body: needs more computing but the results are more accurate
 - Friction: Coulomb/ Shear (Kudo) \rightarrow Bilinear/Arctangent (2nd better)





- Advanced Contact Control: Contact Tolerance Bias Factor can be set: Node can be iterated over the surface – if it falls in the width, the contact type is changed to touching. 0.95 means 5% on the outside of the surface and 95% on the inside. By this, nodes inside the surface (in the tool) can be avoided.
- Analysis Options: If large strain is needed, Updates Lagrange shoul be the computing mode.
- Inside Job: Element type: automatically set, but can be changed.
- If table is set for a material, yield stress should be 1, this is the multiplicator for the table.





Exercise 5 Thin sheet





Exercise 5

Emberi Erőforráso Minisztériuma

- Symmetry on lower edge
- Thickness = 1 mm
- Higher density mesh needed at the stress concentration.

Nemzeti

Tehetség Program









Drawing...











Vaiting

Exercise 6 Sealing assembly





Exercise 6

- Axis symmetry
- Contact bodies, only the Cu sealing is meshed
- From IGS model
























Homework 2&3





Homework 2

- Thickness = 1 mm
- Same as exercise 4, but simulating the whole sheet.
- Compare the results to the results of exercise 4.









Homework 3

- Material: Cu, elastic anf plastic strain
- Same as exercise 4, but simulating the whole sheet.
- Compare the results to the results of exercise 4.







Part 3





Tips and tricks for the exercises of Part 3





Tips and tricks

- If a highly elastic material is used (eg. rubber), type should be set to Mooney
- Expand command can be used in the results file only to the current step – good for visualization
- For symmetry command: 1 point and a normal vector is needed – Sweep after!
- For calculating 2D crack: K1 Toolbox New Crack 2D VCCT – set, select 1 node
 - In Job: select K1, VCCT
 - J-Integral can be also calculated



Tips and tricks

- Axis symmetric simulations: only x axis can be the symmetry axis, geometry can only be drawn in the positive y part of xy plane
- Remesh is better with tri mesh.
- Remeshing: Andvanced
 - Strain change: maximum strain of element before remeshing
 - Distortion: effective maximum strain
 - Penetration: if the elements would overlap, remeshing
 - Parameters: length of element can be set
- Overlapping on purpose: contact table interference fit parameter (drawing to nominal dimensions)





Exercise 7 Sealing assembly (O-ring)





Exercise 7

- Axis symmetry
- Contact bodies, only the rubber O-ring is meshed
- From IGS model
- Only elastic deformation











Exercise 8 Thin sheet with crack





Exercise 8









Exercise 9 Compression





Exercise 9

- Compression between flat plates
- Al 2024 AlCuMg2
- Room temperature: $k_f = 336 \cdot \varphi^{0.1547}$
- Remeshing

























Part 4





Tips and tricks for the exercises of Part 4





Tips and tricks

- Welding in 2D 1 section
- ToolBox Weld Path
- Loadcase Solution control
 - Non positive definite if the computing stops due to the DoF-s, this lets through
 - Pay attention to wrong result nodes the computing can stop due to them
- Transform into 3D expand almost everything can be transformed





Exercise 10 Welding in 2D





Exercise 10

- 2D welding 1 section
- Thermal+structural job
- Without welding filler metal







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Read

Exercise 11&12 Welding in 3D





Exercise 11

Emberi Erőforráso Minisztériuma

- 3D welding whole process
- Thermal+structural job
- Without welding filler metal
- Temperature tracking during welding



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Exercise 12

- 3D welding whole process
- Thermal+structural job
- Without welding filler metal
- Plastic deformation tracking during and after welding

Nemzeti





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Part 5





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Exercise 13-16 Beam vibration





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Exercise 13

- Fixed x=0 and z=0 displacement on left side
- y=10 amplitude on left side at 10 Hz frequency
- Dynamic transient loadcase
- Only elastic deformation, 2D plane strain







Exercise 14

- Fixed x=0, y=0 and z=0 displacement on left side
- Dynamic modal analysis, eigenfrequencies
- 2D plane strain















Exercise 15

- Fixed x=0, y=0 and z=0 displacement on left side
- Dynamic modal analysis, eigenfrequencies
- 3D simulation


























Exercise 16

- Fixed x=0 and z=0 displacement on left side
- y=10 amplitude on left side at 61 Hz frequency
- Dynamic transient loadcase
- Only elastic deformation, 2D plane strain







Eigenfrequencies

- 2D vs 3D
- In 2D, some eigenfrequencies are missing

| | 2D (Hz) | 3D (Hz) |
|----|---------------|---------|
| 1 | 10.0 | 9.8 |
| 2 | - | 41.5 |
| 3 | 63.0 | 61.5 |
| 4 | 176.4 | 172.0 |
| 5 | - | 257.5 |
| 6 | - | 299.1 |
| 7 | 345.4 | 336.9 |
| 8 | 570.5 | 556.6 |
| 9 | - | 708.6 |
| 10 | not simulated | 831.0 |







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